

Equations for the exam

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- • Bernoulli equation:

$$\frac{p_1}{\rho g} + z_1 + \frac{v_1^2}{2g} = \frac{p_2}{\rho g} + z_2 + \frac{v_2^2}{2g} + h_{f1 \rightarrow 2}$$

For channels:

$$y_1 + z_1 + \frac{v_1^2}{2g} = y_2 + z_2 + \frac{v_2^2}{2g} + h_{f1 \rightarrow 2}$$

- Local (Minor) losses:

$$h_m = K \frac{v^2}{2g} ; K \text{ from tables}$$

- Manning's equation:

$$V = \frac{K_m}{m} R^{2/3} S_o^{1/2} ; K_m = .1 \text{ in SI}$$

$K_m = 1.486 \text{ in English units.}$

- Chezy equation:

$$V = C R^{1/2} S_o^{1/2}$$

- Shear stress at bottom of channel:

$$\tau_w = \rho g R S_o$$

- • $Re = \frac{VD}{\nu}$

- Direct method by Rouse

$$B = hf g D^3 / (L v^2)$$

(dimensionless)

$$Re = - (8B)^{1/2} \log_{10} \left[\frac{\epsilon/D}{3.7} + \frac{1.775}{\sqrt{B}} \right]$$

- Darcy - Weisbach equation:

$$hf = f \frac{L}{D} \frac{v^2}{2g}$$

- Froude number

$$Fr = \frac{v}{\sqrt{g y}} \leftarrow \text{or } L \text{ in general}$$

- Critical depth for rectangular cross section

$$y_c = \sqrt[3]{\frac{q^2}{g}}$$

$$R = \frac{\text{Area}}{\text{Wetted perimeter}}$$

- Gradually varied flow

$$\frac{dy}{dx} = \frac{S_0 - S_f}{1 - Fr^2}$$

- Hydraulic jump

$$\frac{y_2}{y_1} = \frac{1}{2} \left(\sqrt{1 + 8Fr_1^2} - 1 \right)$$

$$\Delta E = \frac{(y_2 - y_1)^3}{4 y_2 y_1}$$

- Colebrook-White equation

$$\frac{1}{\sqrt{f}} = -2 \log_{10} \left(\frac{\epsilon/D}{3.7} + \frac{2.51}{Re \sqrt{f}} \right)$$

- Iterative method to compute Q given D, L, ϵ , f , μ , h_f

$$f V^2 = \frac{h_f D 2g}{L} = B = \text{known} \quad \text{dimensional}$$

- POWER:

$$\text{Power} = \rho g h_{\text{pump}} Q$$

- Power required = $\frac{\text{Power delivered to water}}{\eta}$

- Explicit formula:

$$\frac{1}{f^{1/2}} \approx -1.8 \log \left[\frac{6.9}{Re} + \left(\frac{\epsilon/D}{3.7} \right)^{1.11} \right]$$

- Discharge of broad-crested weir

$$C_{wb} = \frac{0.65}{(1 + H/p)^{1/2}}$$

$$Q = C_{wb} b \sqrt{g} \left(\frac{2}{3}\right)^{3/2} H^{3/2}$$

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- Discharge of V-shaped weir:

$$Q = 0.7542 \times g^{1/2} \times \tan \alpha \times H^{5/2}$$

- Conversions:

$$1 \text{ gallon} = 3.785 \text{ l}$$

$$1 \text{ hp} = 745.7 \frac{\text{Nm}}{\text{s}}$$

$$1 \text{ hp} = 550 \frac{\text{lb ft}}{\text{s}}$$

- Drag (skin formulas):

$$F_{\text{drag}} = \frac{1}{2} C_D \rho U_{\infty}^2 A$$

$$C_D = \frac{0.031}{(\text{Re}_L)^{1/7}} ;$$

if turbulent ($\text{Re}_L > 3 \times 10^6$)

$$C_D = \frac{1.328}{(\text{Re}_L)^{1/2}}$$

- Reynolds scaling:

$$\frac{V_M}{V_P} = \frac{L_P}{L_M} = \lambda^{-1}$$

- Froude scaling:

$$\frac{V_M}{V_P} = \lambda^{1/2}$$

- Specific speed of pumps

$$N_s = \frac{\omega \sqrt{Q}}{(gH)^{3/4}}$$

- Net positive suction head:

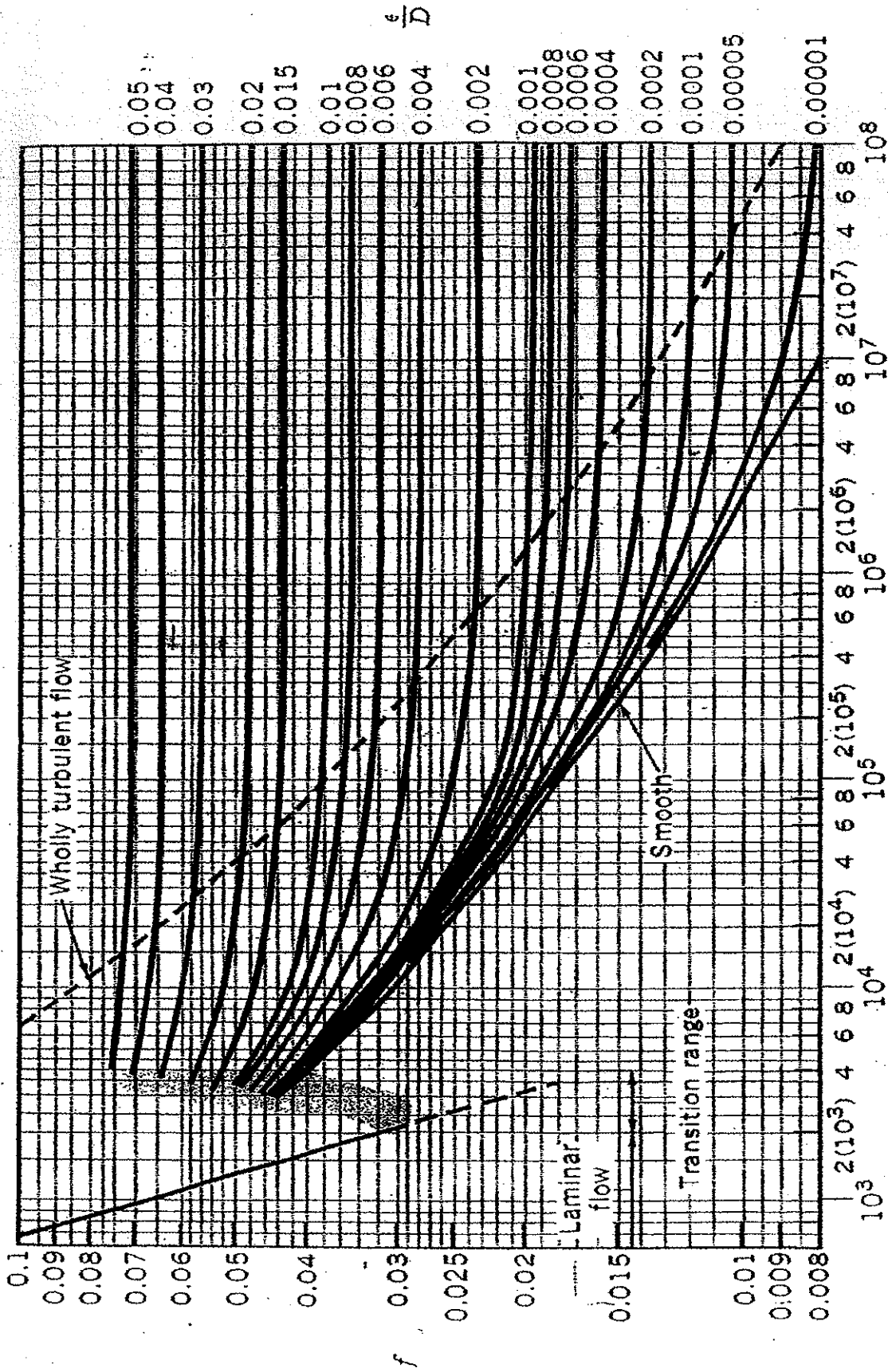
$$NPSH = \frac{P_{inlet}}{\gamma} - \frac{P_{vapor}}{\gamma} =$$

$$\frac{P_{atm}}{\gamma} - \left[H_s + h_{f_{1 \rightarrow 3}} + \sum h_{m_{1 \rightarrow 3}} + \frac{V_{inlet}^2}{2g} + \frac{P_{vapor}}{\gamma} \right]$$

- $P_{vapor\ 20^\circ C} = 2.337 \text{ kPa}$

$$P_{vapor\ 30^\circ C} = 4.242 \text{ kPa}$$

$$P_{atmospheric} = 101,350 \text{ Pa} \approx 10^5 \text{ Pa}$$



$$Re = \frac{\rho V D}{\mu}$$

■ FIGURE 8.23 Friction factor as a function of Reynolds number and relative roughness for round pipes—the Moody chart (Data from Ref. 7 with permission).

port dashed or dotted lines.

known backwater curve; it is in the practical point of view. end of a long mild channel is than the normal depth of the in zone I. The upstream end depth line, since $dy/dx = 0$ as to the horizontal pool surface, profiles of the M1 profile are the Fig. 9-4a) and the profile in a

of the channel at the down-a depth less than the normal profile is tangent to the normal- f the amount of submergence ical depth, the flow profile will a vertical line at a depth equal for $y = y_c$. This means the h of submergence at the down- th, then as much of the profile n the reservoir. Examples are an enlargement of a canal cross al leading to a reservoir, where l below the critical-depth line

upstream channel bottom, e angle, depending on the type and terminates with a hydraulic of profile usually occurs when a

The beginning of the profile, by the theory, depends on the e higher the velocity, the farther theoretical upstream end of the u. At this end $y = 0$; thus the re, the theoretical upstream end ly. Examples of the M3 profile (Fig. 9-4e) and the profile after o mild (Fig. 9-4f).

ie upstream and becomes tangent stream end. Examples are the

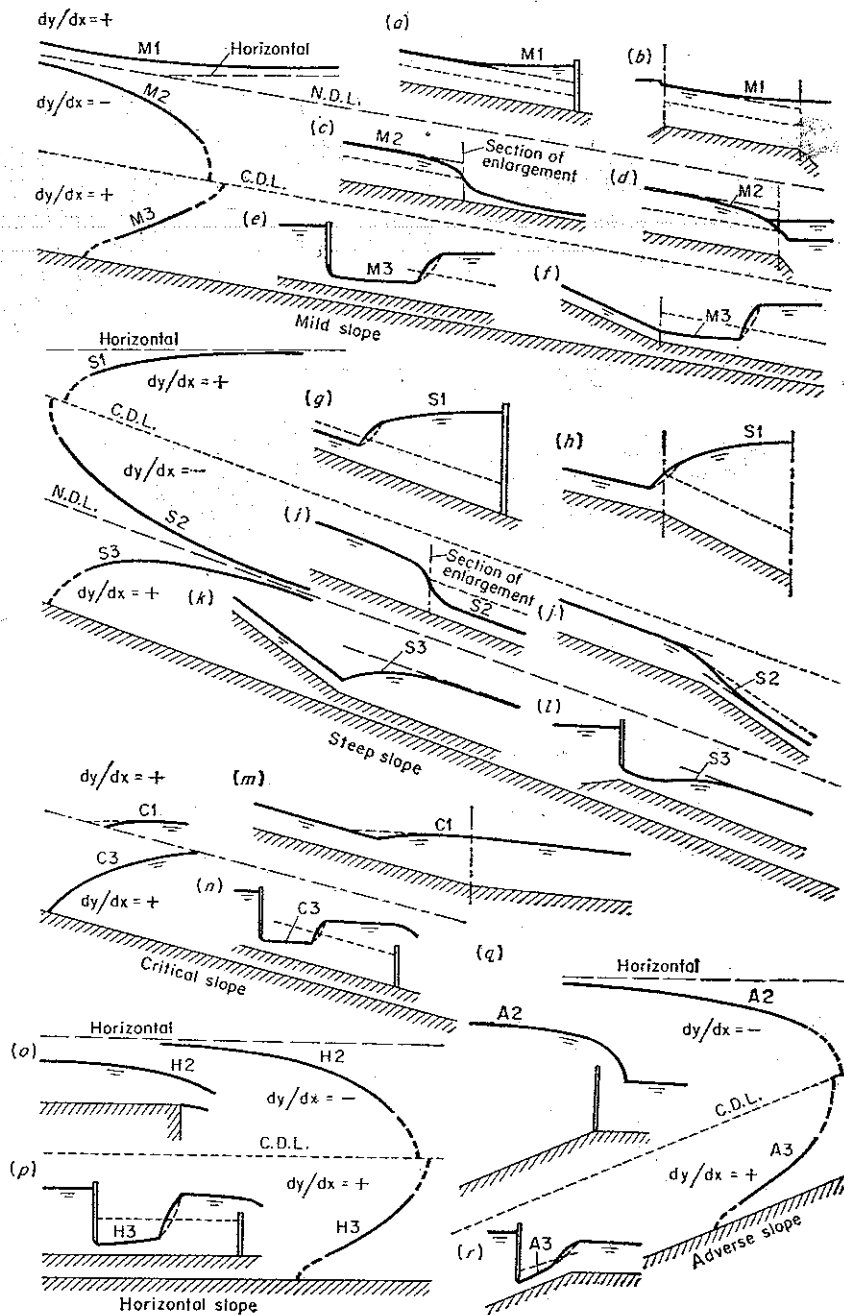


FIG. 9-4. Examples of flow profiles.